Q1. A Little Unit of Feedback  [10 %]
Consider a unity, negative feedback systems with the following open loop transfer function:

\[ G(s) = \frac{10}{s+1} \]

Determine the output \( y(t) \) for the following reference inputs:

a. \( r(t) = \sin(t) \)
b. \( r(t) = \cos(t) \)
c. \( r(t) = \sin(t) - \cos(t) \)
d. \( r(t) = \sin(t) + \cos(t) \)

Hints:
- Note that the reference inputs, \( r(t) \), and requested outputs, \( y(t) \) are in the time domain, but that the plant is in the s-domain. It might help to convert \( r(t) \) to the Laplace domain first.
- The feedback path (often indicated as \( H(s) \)) is unity
- In determining the solutions to parts (b) through (d), you may reference to solutions from previous part (a) through (c). However, answers have to be clear and justified.

⇒ [ Updates (19/5):]
- Added in ‘negative feedback’ to clarify type of feedback
Q2. Concepts: Loop Gain

Consider the following closed-loop, negative feedback controller in the s-domain where \( C(s) \) is the controller, \( G(s) \) is the plant, \( H(s) \) is the sensor, \( R(s) \) is the reference signal, \( E(s) \) is the error, \( TE(s) \) is the tracking error signal \( TE(s) = R(s) - Y(s) \) (not shown), \( D(s) \) is the disturbance signal and \( N(s) \) is sensing noise.

For such a system, we can define the **Loop Gain**, \( L(s) \):

\[
L(s) = C(s)G(s)H(s)
\]

In this, please consider and discuss:

a. For the unity feedback case (**where** \( H(s) = 1 \)), show that the tracking error, which we define as \( TE(s) = R(s) - Y(s) \) (not to be confused with \( E(s) \)), is:

\[
TE(s) = \frac{R(s)}{1 + L(s)} - \frac{D(s)G(s)}{1 + L(s)} + \frac{L(s)N(s)}{1 + L(s)}
\]

b. Does a large loop gain, \( L(s) \), increase or reduce the influence of a disturbance, \( D(s) \), to the tracking error (\( TE(s) \))? Why?

c. Does a large loop gain, \( L(s) \), increase or reduce the influence of noise, \( N(s) \), to the tracking error (\( TE(s) \))? Why?

d. Reflecting on the results from above, how does this suggest we should shape \( L(s) \) as a function of frequency? Why?

Please be analytic, specific and clear in your explanation. Waffling will not be accepted.

⇒ [ Update (18/5): Tracking error (\( TE(s) \)) notation added to help clarify this and note it as distinct from the feedback loop error (\( E(s) \)). Even if one solves for \( E(s) \) the trends being considered remain the same ]
Q3. Mind The Step?

Please consider the steady state error ($e_{ss}$) of a negative, unity feedback control process under the following inputs $R(t)$ (step, ramp, and parabola) for Type 0, I, and II plants and output $Y(t)$.

Thus, the Question: With this in mind, please determine the following:

a. For a step input of magnitude $K$, determine the steady state error for a Type 0, I, and II system. (Please remember to show your working)

b. For a ramp input of magnitude $K$, determine the steady state error for a Type 0, I, and II system. (Please remember to show your working)

c. For a parabolic input of magnitude $K$, determine the steady state error for a Type 0, I, and II system. (Please remember to show your working)

d. Using the results from above, please present a summary table of steady-state errors such as:

<table>
<thead>
<tr>
<th>System Type</th>
<th>Input Type</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td>Step</td>
</tr>
<tr>
<td>0</td>
<td></td>
</tr>
<tr>
<td>I</td>
<td></td>
</tr>
<tr>
<td>II</td>
<td></td>
</tr>
</tbody>
</table>

Hints:

- More so than the answer to the table in Part (d), we are interested in your reasoning and process. Be sure to justify your answer.
- $e_{ss} = \lim_{t \to \infty} e(t) = \lim_{s \to 0} sE(s)$ (as per the final value theorem)

Note that:

- A Type $n$ plant refers to the number of integrators in $G(s)$ (eg, the system $G(s) = \frac{a}{s^2b}$ is a Type 0 plant\(^1\), but $G(s) = \frac{a}{s(s^2+b)}$ is a Type 1 plant)
- A step input of magnitude $K$ may be considered as $r(t) = K$ or $r(s) = \frac{K}{s}$
- A ramp input of magnitude $K$ may be considered as $r(t) = Kt$ or $r(s) = \frac{K}{s^2}$
- A parabolic input of magnitude $K$ may be considered as $r(t) = \frac{Kt^2}{2}$ or $r(s) = \frac{K}{s^3}$
- For convenience we may define the following steady state errors
  - $P$: steady state position error, $P = \lim_{s \to 0} G(s)$
  - $V$: steady state velocity error, $V = \lim_{s \to 0} sG(s)$
  - $A$: steady state acceleration error, $A = \lim_{s \to 0} s^2 G(s)$

\[\Rightarrow\text{ Updates (19/5):}\]

- Fixed typo in the first sentence. Please solve for Type 0, 1, & 2 Plants
- Fixed Typo - should be $G(s)$ (Note 1 said $C(s)$)
- Clarified plant type by providing example of Type 0 and I plants.
- Clarified notation to prevent confusion with Question 2 (which is different) \(\Rightarrow\]

\(^1\) The system type is different to system order. A system is called Type 0, I, II, etc. if $N=0,1,2,...$ respectively as given in $G(s)=[K(T_a\cdot s+1)(T_b\cdot s+1)...(T_m\cdot s+1)] / [s^N(T_1\cdot s +1)(T_2\cdot s+1)...(T_p\cdot s+1)]$
Q4. PID’s New PID?
Universal Questocopter (UniQuest) seeks your advice on whether they should patent and license the following control circuit presented to them by Dr Ahab of Pequod Industrial Design. He claims to have invented a new PID controller using an integrator and a first-order lag controller as follows:

Is Dr Ahab correct? (i.e. is this a “lead-lag compensator”)? Should UniQuest patent this?

Let’s investigate this by determining the transfer function \( C(s) = \frac{E_{out}(s)}{E_{in}(s)} \) for this circuit

a. What is the impedance of \( Z_1 \) and \( Z_2 \)?
b. What is \( \frac{E(s)}{E_{in}(s)} \)?
c. What is \( \frac{E_{out}(s)}{E(s)} \)?
d. Now consequently what is the transfer function for the entire system as a function of \( R_1, R_2, R_3, R_4, R_5, C_1 \) and \( C_2 \)?
e. It’s claimed that ratio \( \frac{R_5}{R_4} \) adjusts the overall gain of the controller. Is this correct?
In 1942, Ziegler and Nichols empirically derived two methods to help tune PID controllers:

- The transient-response [aka “first method” or “open-loop Ziegler and Nichols”], and
- The stability-limit [aka “second method” or “closed-loop Ziegler and Nichols”].

Of this, let us consider the performance of the Transient-Response Method for a standard negative feedback control case where $C(s) = K_P(1 + \frac{1}{T_I s} + T_D s)$:

Recall that the first method is particularly apt for systems that can be approximated as a first-order plant having a transport delay ($L$) and rise slope ($K/\tau$), thus $G(s) = \frac{K e^{-Ls}}{s \tau + 1}$

**Step I: Simulation**

Let us simulate the influence of the parameters $K$, $L$, and $\tau$. For $K=\{1, 5, 10 and 100\}$, $L=\{1, 5, 10 and 100\}$ and $\tau=\{1, 5, 10 and 100\}$, please write a program to:

1. Determine $K_P$, $T_I$, and $T_D$ (as needed) for each of a P, PI and PID controller; and
2. Plot the output response ($\frac{Y(s)}{R(s)}$) of each of these three case to a unit step.

For $K=10$ and $L=\{The last digit of your student number\}+1$, present your results against the variations in $\tau$. Notably, please present:

- (a) the resulting values for $K_P$, $T_I$ and $T_D$ in this case
- (b) P, PI, and PID step response plots (against time)

**Step II: Analysis**

Clearly, the Ziegler-Nichols method will not work with all plants and processes. Using the results from the simulation (Step I), fully discuss the nature and limitations of this approach (in other words where does it work well? and where does it fail?).

Some points to consider:

- The “first method” requires taking the controller ($C(s)$) out-of-the-loop (hence the term “open-loop”). Discuss the benefits and disadvantages of this.
- The average decay ratio (in general, on the whole), is it the $\frac{1}{4}$ decay expected?
- What would happen if there is measurement noise?
- Applicability of this approach to processes with a large $L/\tau$ (eg, $L/\tau=10$)
- Applicability of this approach to processes with multiple latencies/time lags

- [Optional – though required for a High-Distinction mark on this question]
  Considering that Ziegler-Nichols was designed to provide the best disturbance rejection instead of the best input response performance, describe an “improved Ziegler-Nichols” method that would focus on the latter (i.e., step response).

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Attaching commented code:
Please attach your fully documented code that can generate the values and plots for the general simulation case (ie, K=[1, 5, 10, 100], L=[1, 5, 10, 100], τ=[1, 5, 10, 100]). It is expected that the code will be written to a “professional standard”.

Hints:

- The program may be in any language of your choosing, but the tutors typically support MATLAB.
- Consider the Matlab step command documentation.
- In (b) please show time on x-axis and normalized magnitude on the y-axis.
- One way to present part (b) is to have 3 plots (one each for the P, PI, and PID control architecture) and each plot would show 4 step responses (for one values of τ=[1, 5, 10 and 100]). Another way to show this is to have 4 plots (one each for the values of τ) with each plot in this case showing 3 responses for the (P, PI, and PID architectures).
- In general simulation the code will generate a considerable amount of results. We are only asking for a subset of these results to be presented. Please do not plot the results for all cases; kindly only the cases requested in Step I or those you wish to include as part your explanation in Step II.